

LECTURE 15: SOME APPLICATIONS OF SECOND ORDER DE'S  
HIGHER ORDER DIFFERENTIAL EQUATIONS (VII)

(Text: Chap. 4)

## 1 (\*) Vibration System

We now give an application of the theory of second order DE's to the description of the motion of a simple mass-spring mechanical system with a damping device. We assume that the damping force is proportional to the velocity of the mass. If there are no external forces we obtain the differential equation

$$m \frac{d^2x}{dt^2} + b \frac{dx}{dt} + kx = 0$$

where  $x = x(t)$  is the displacement from equilibrium at time  $t$  of the mass of  $m > 0$  units,  $b \geq 0$  is the damping constant and  $k > 0$  is the spring constant. In operator form with  $D = \frac{d}{dt}$  this DE is, after normalizing,

$$\left( D^2 + \frac{b}{m}D + \frac{k}{m} \right) (x) = 0.$$

The characteristic polynomial  $r^2 + (b/m)r + k/m$  has discriminant

$$\Delta = (b^2 - 4km)/m^2.$$

If  $b^2 < 4km$  we have  $\Delta < 0$  and the characteristic polynomial factorizes in the form  $(r + b/2m)^2 + \omega^2$  with

$$\omega = \sqrt{4km - b^2}/2m = \sqrt{\frac{k}{m} - (b/2m)^2}.$$

In this case the characteristic polynomial has complex roots  $-b/2m \pm i\omega$  and the general solution of the DE is

$$y = e^{-bt/2m}(A \cos(\omega t) + B \sin(\omega t)) = Ce^{-bt/2m} \sin(\omega t + \theta)$$

where  $C = \sqrt{A^2 + B^2}$  and  $0 \leq \theta \leq 2\pi$  defined by  $\cos(\theta) = A/C$ ,  $\sin(\theta) = B/C$ . The angle  $\theta$  is called the **phase**. In this case we see that the mass oscillates with **frequency**  $\omega/2\pi$  and decreasing amplitude. If  $b = 0$  there is no damping and the mass oscillates with frequency  $\omega/2\pi$  and constant amplitude; such motion is called **simple harmonic**.

If  $b^2 \geq 4km$  we have  $\Delta \geq 0$  and so the characteristic polynomial has real roots

$$r_1 = -b/2m + \sqrt{b^2 - 4km}/2m, \quad r_2 = -b/2m - \sqrt{b^2 - 4km}/2m$$

which are both negative. If  $r_1 = r_2 = r$  the general solution of our DE is

$$y = Ae^{rt} + Bte^{rt}$$

and if  $r_1 \neq r_2$  the general solution is

$$y = Ae^{r_1 t} + Be^{r_2 t}.$$

In both cases  $y \rightarrow 0$  as  $t \rightarrow \infty$ . In the second case we have what is called **over damping** and in the first case the over damping is said to be **critical**. In each the mass returns to its equilibrium position without oscillations.

Suppose now that our mass-spring system is subject to an external force so that our DE now becomes

$$m \frac{d^2x}{dt^2} + b \frac{dx}{dt} + kx = F(t).$$

The function  $F(t)$  is called the **forcing function** and measures the magnitude and direction of the external force. We consider the important special case where the forcing function is harmonic

$$F(t) = F_0 \cos(\gamma t), \quad F_0 > 0 \text{ a constant.}$$

We also assume that we have under-damping with damping constant  $b > 0$ . In this case the DE has a particular solution of the form

$$y_p = A_1 \cos(\gamma t) + A_2 \sin(\gamma t).$$

Substituting the the DE and simplifying, we get

$$((k - m\gamma^2)A_1 + b\gamma A_2) \cos(\gamma t) + (-b\gamma A_1 + (k - m\gamma^2)A_2) \sin(\gamma t) = F_0 \cos(\gamma t).$$

Setting the corresponding coefficients on both sides equal, we get

$$\begin{aligned} (k - m\gamma^2)A_1 + b\gamma A_2 &= F_0, \\ -b\gamma A_1 + (k - m\gamma^2)A_2 &= 0. \end{aligned}$$

Solving for  $A_1, A_2$  we get

$$A_1 = \frac{F_0(k - m\gamma^2)}{(k - m\gamma^2)^2 + b^2\gamma^2}, \quad A_2 = \frac{F_0 b\gamma}{(k - m\gamma^2)^2 + b^2\gamma^2}$$

and

$$\begin{aligned} y_p &= \frac{F_0}{(k - m\gamma^2)^2 + b^2\gamma^2} ((k - m\gamma^2) \cos(\gamma t) + b\gamma \sin(\gamma t)) \\ &= \frac{F_0}{\sqrt{(k - m\gamma^2)^2 + b^2\gamma^2}} \sin(\gamma t + \phi). \end{aligned}$$

The general solution of our DE is then

$$y = C e^{-bt/2m} \sin(\omega t + \theta) + \frac{F_0}{\sqrt{(k - m\gamma^2)^2 + b^2\gamma^2}} \sin(\gamma t + \phi).$$

Because of damping the first term tends to zero and is called the **transient** part of the solution. The second term, the **steady-state** part of the solution, is due to the presence of the forcing function  $F_0 \cos(\gamma t)$ . It is harmonic with the same frequency  $\gamma/2\pi$  but is out of phase with it by an angle  $\phi - \pi/2$ . The ratio of the magnitudes

$$M(\gamma) = \frac{1}{\sqrt{(k - m\gamma^2)^2 + b^2\gamma^2}}$$

is called the **gain** factor. The graph of the function  $M(\gamma)$  is called the **resonance curve**. It has a maximum of

$$\frac{1}{b\sqrt{\frac{k}{m} - \frac{b^2}{4m^2}}}$$

when  $\gamma = \gamma_r = \sqrt{\frac{k}{m} - \frac{b^2}{4m^2}}$ . The frequency  $\gamma_r/2\pi$  is called the **resonance frequency** of the system. When  $\gamma = \gamma_r$  the system is said to be in resonance with the external force. Note that  $M(\gamma_r)$  gets arbitrarily large as  $b \rightarrow 0$ . We thus see that the system is subject to large oscillations if the damping constant is very small and the forcing function has a frequency near the resonance frequency of the system.

The above applies to a simple LRC electrical circuit where the differential equation for the current  $I$  is

$$L \frac{d^2 I}{dt^2} + R \frac{dI}{dt} + I/C = F(t)$$

where  $L$  is the inductance,  $R$  is the resistance and  $C$  is the capacitance. The resonance phenomenon is the reason why we can send and receive and amplify radio transmissions sent simultaneously but with different frequencies.